NEAR-WALL ENSTROPHY GENERATION IN A DRAG-REDUCED TURBULENT CHANNEL FLOW WITH SPANWISE WALL **OSCILLATIONS**

Pierre Ricco¹, Claudio Ottonelli, Yosuke Hasegawa, Maurizio Quadrio 1 Sheffield, 2 Onera Paris,

3 Tokyo, 4 Politecnico Milano

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Energy input into system

Pre-determined forcing

NUMERICAL APPROACH

Direct numerical simulations of wall turbulence Fully-developed turbulent channel flow ($Re_{\tau} = u_{\tau}h/\nu = 200$ Compact finite-difference scheme along wall-normal direction Spectral discretization along streamwise and spanwise direct

SPANWISE WALL OSCILLATIONS

- New approach: Turbulent enstrophy
- Transient evolution

CONSTANT DP/DX

 au_w is fixed in fully-developed conditions

GAIN: *U_b* increases

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$$\overline{f}(y,t) = \frac{1}{L_x L_z} \int_0^{L_x} \int_0^{L_z} f(x,y,z,t) \mathrm{d}z \mathrm{d}x$$

PHASE $\widehat{f}(y,\tau) = \frac{1}{N} \sum_{n=0}^{N-1} \overline{f}(y,nT+\tau)$

TIME $\left\langle f \right\rangle (y) = rac{1}{T} \int_{0}^{T} f(y, au) \mathrm{d} au$

GLOBAL

$$[f]_{g} = \int_{0}^{h} \langle f \rangle (y) \mathrm{d}y$$

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Scaling by viscous units

Mean velocity increases in the bulk of the channel

Mean wall-shear stress is unchanged



Scaling by viscous units

Mean velocity increases in the bulk of the channel Mean wall-shear stress is unchanged Optimum period of oscillation $T \approx 75$



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Turbulence kinetic energy decreases

Streamwise velocity fluctuations are attenuated the most New oscillatory Reynolds stress term $\widehat{\mu}$ is created $\widehat{/\mu}$



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Streamwise velocity fluctuations are attenuated the most New oscillatory Reynolds stress term \widehat{vw} is created, $\langle \widehat{vw} \rangle = 0$



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Energy is fed through $P_x (\rightarrow U_b \tau_w)$ and wall motion $(\rightarrow \mathcal{E}_w)$ Energy is dissipated through: Mean-flow viscous effects (streamwise $\rightarrow \mathcal{D}_U$, spanwise $\rightarrow \mathcal{D}_W$) Turbulent viscous effects $(\rightarrow \mathcal{D}_T)$



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ENERGY BALANCE: EQUATIONS



GLOBAL TURBULENT KINETIC ENERGY EQUATION

$$\underbrace{\left[\widehat{uv}\frac{\partial\widehat{U}}{\partial y}\right]_{g}}_{\mathcal{P}_{uv}} + \underbrace{\left[\widehat{vw}\frac{\partial\widehat{W}}{\partial y}\right]_{g}}_{\mathcal{P}_{vv}} + \left[\frac{\partial\widehat{u_{i}}\frac{\partial\overline{u_{i}}}{\partial x_{j}}\frac{\partial\overline{u_{i}}}{\partial x_{j}}\right]_{g} = 0$$

FOTAL KINETIC ENERGY BALANCE

$$U_b \tau_w + \mathcal{E}_w = \mathcal{D}_U + \mathcal{D}_W + \mathcal{D}_T$$

FURBULENT DISSIPATION

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$$\mathcal{D}_{\mathcal{T}} = \left[\widehat{\omega_i \omega_i}\right]_g$$

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GLOBAL MEAN KINETIC ENERGY EQUATION

$$U_{b}\tau_{W} + \underbrace{\left\langle A \frac{\partial \widehat{W}}{\partial y} \Big|_{y=0} \right\rangle}_{\mathcal{E}_{W}} = -\underbrace{\left[\widehat{u}\widehat{v}\frac{\partial \widehat{U}}{\partial y}\right]_{g}}_{\mathcal{P}_{UV}} - \underbrace{\left[\widehat{v}\widehat{w}\frac{\partial \widehat{W}}{\partial y}\right]_{g}}_{\mathcal{P}_{VW}} + \underbrace{\left[\left(\frac{\partial \widehat{U}}{\partial y}\right)^{2}\right]_{g}}_{\mathcal{D}_{U}} + \underbrace{\left[\left(\frac{\partial \widehat{W}}{\partial y}\right)^{2}\right]_{g}}_{\mathcal{D}_{W}}$$

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TURBULENT DISSIPATION

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Why does TKE decrease?

THREE POSSIBILITIES

Stokes layer acts on \mathcal{D}_U directly

 \rightarrow excluded because W does not work directly on $(\partial U/\partial y)^2$

- Stokes layer acts on P_{uv} directly
 - \rightarrow excluded because W does not work directly on uv
- Stokes layer acts on $\mathcal{D}_{\mathcal{T}} = \left[\widehat{\omega_i \omega_i}\right]_q$ directly
 - ightarrow W works on turbulent vorticity transport

TURBULENT ENSTROPHY TRANSPORT

Study the transport of turbulent enstrophy $\widehat{\omega}_i \widehat{\omega}_i$

The term *enstrophy* was coined by G. Nickel and is from Greek $\sigma \tau \rho o \phi \eta$, which means *turn*

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Stokes layer influences dynamics of turbulent enstrophy Three terms: which is the dominating one?

 \rightarrow Let's look at the terms of the equation

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$$\frac{1}{2} \frac{\partial \widehat{\omega_{i}\omega_{i}}}{\partial \tau} = \underbrace{\widehat{\omega_{x}\omega_{y}}}_{2} \frac{\partial \widehat{U}}{\partial y} + \underbrace{\widehat{\omega_{z}\omega_{y}}}_{3} \frac{\partial \widehat{W}}{\partial y} + \underbrace{\widehat{\omega_{j}}}_{4} \underbrace{\frac{\partial \widehat{W}}{\partial x_{j}}}_{4} - \underbrace{\widehat{\omega_{j}}}_{5} \frac{\partial \widehat{W}}{\partial x_{j}} \frac{\partial \widehat{U}}{\partial y} \\ - \underbrace{\underbrace{\widehat{V}\omega_{x}}}_{6} \frac{\partial^{2} \widehat{W}}{\partial y^{2}} + \underbrace{\widehat{V}\omega_{z}}_{7} \frac{\partial^{2} \widehat{U}}{\partial y^{2}} + \underbrace{\widehat{\omega_{i}\omega_{j}}}_{8} - \frac{1}{2} \frac{\partial}{\partial y} \left(\underbrace{V\omega_{i}\omega_{i}}_{9} \right) \\ + \frac{1}{2} \frac{\partial^{2} \widehat{\omega_{i}\omega_{i}}}{\partial y^{2}} - \underbrace{\frac{\partial \widehat{\omega_{i}}}{\partial x_{j}} \frac{\partial \omega_{i}}{\partial x_{j}}}_{11} .$$

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TURBULENT ENSTROPHY PROFILES FIXED WALL



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Term 3, $\widehat{\omega_z \omega_y} \partial W / \partial y \rightarrow \underline{\text{turbulent enstrophy production}}$ is dominant Other oscillating-wall terms are smaller Turbulent dissipation of turbulent enstrophy increases

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We have not answered questions on TKE and U_b , yet

Key: transient from start-up of wall motion



USEFUL INFORMATION

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USEFUL INFORMATION

RED: term 3 increases abruptly, then decreases

We have not answered questions on TKE and U_b , yet

Key: transient from start-up of wall motion



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BLACK: turbulent enstrophy increases, then decreases

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USEFUL INFORMATION

RED: term 3 increases abruptly, then decreases

BLACK: turbulent enstrophy increases, then decreases

BLUE: TKE decreases monotonically

Turbulent enstrophy increases through $\widehat{\omega_z \omega_y} \partial W / \partial y$

INTERMEDIATE STAGE

TKE decreases because of enhanced turbulent dissipation

Long stage

Bulk velocity increases because of TKE reduction

→ drag reduction

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5 DECEMBER 2012 WALL-OSCILLATION DRAG-REDUCTION PROBLEM 18-1

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Initial state





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PHYSICAL INTERPRETATION OF $\widehat{\omega_z \omega_y} \partial \widehat{W} / \partial y$

- $\widehat{\omega_z \omega_y} \partial \widehat{W} / \partial y$ is key term leading to drag reduction
- $\widehat{\omega_z \omega_y} \partial \widehat{W} / \partial y \to \partial \widehat{W} / \partial y$ acts on $\widehat{\omega_z \omega_y}$
- $\widehat{\omega_z \omega_y} \approx \frac{\partial u}{\partial y} \frac{\partial u}{\partial z}$ $\frac{\partial u}{\partial y} \rightarrow$ upward eruption of near-wall low-speed fluid $\frac{\partial u}{\partial z} \rightarrow$ lateral flanks of the low-speed streaks



$\frac{\partial u}{\partial y} \frac{\partial u}{\partial z}$ located at the sides of high-speed streaks

MODELLING TURBULENT ENSTROPHY PRODUCTION



SIMPLIFIED TURBULENT ENSTROPHY EQUATION

$$\frac{1}{2}\frac{\partial}{\partial t}\left(\omega_{y}^{2}+\omega_{z}^{2}\right)=\omega_{z}\omega_{y}G-\left(\frac{\partial\omega_{y}}{\partial y}\right)^{2}-\left(\frac{\partial\omega_{z}}{\partial y}\right)^{2}$$

Rotation of axis

$$\frac{1}{2}\frac{\partial\omega_n^2}{\partial t} = S_{nn}\omega_n^2 - \left(\frac{\partial\omega_n}{\partial y}\right)^2$$

Integration by Charpit's method

$$\omega_n = \omega_{n,0} \underbrace{\mathrm{e}^{\sin \alpha \cos \alpha Gt}}_{\text{stretching}} \underbrace{\mathrm{e}^{-\beta^2 t} \mathrm{e}^{-\beta y}}_{\text{dissination}}, \beta = \frac{\partial \omega_n / \partial t}{\partial \omega_n / \partial y} \sim \frac{\lambda_y}{\lambda_t}$$

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OSCILLATION PERIOD VS. TERM 3



Drag reduction grows monotonically with global production term This happens up to optimum period

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THANK YOU!

REFERENCE

Ricco, P. Ottonelli, C. Hasegawa, Y. Quadrio, M. Changes in turbulent dissipation in a channel flow with oscillating walls *J. Fluid Mech.*, 700, 77-104, 2012.

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